



Provided by the author(s) and University of Galway in accordance with publisher policies. Please cite the published version when available.

Title	Probing the low-temperature chemistry of ethanol via the addition of dimethyl ether
Author(s)	Zhang, Yingjia; El-Merhubi, Hilal; Lefort, Benoîte; Le Moyne, Luis; Curran, Henry J.; Kéromnès, Alan
Publication Date	2017-11-28
Publication Information	Zhang, Yingjia, El-Merhubi, Hilal, Lefort, Benoîte, Le Moyne, Luis, Curran, Henry J., & Kéromnès, Alan. (2018). Probing the low-temperature chemistry of ethanol via the addition of dimethyl ether. <i>Combustion and Flame</i> , 190, 74-86. doi: 10.1016/j.combustflame.2017.11.011
Publisher	Elsevier
Link to publisher's version	https://doi.org/10.1016/j.combustflame.2017.11.011
Item record	http://hdl.handle.net/10379/14790
DOI	http://dx.doi.org/10.1016/j.combustflame.2017.11.011

Downloaded 2024-05-05T15:55:07Z

Some rights reserved. For more information, please see the item record link above.



1 An Experimental and Kinetic Modeling Study of Ethanol/DME Mixtures Auto-Ignition

2 Y. Zhang^{1,2}, H. El-Merhubi³, B. Lefort³, L. Le Moyne³, H.J. Curran¹, A. Kéromnès^{3*}

3 ¹Combustion Chemistry Centre, National University of Ireland Galway, Ireland

4 ²State Key Laboratory of Multiphase Flow in Power Engineering, Xi'an Jiaotong University, Xi'an 710049, China

5 ³DRIVE EA1859, Univ. Bourgogne Franche Comté, F58000 Nevers, France

6 Abstract

7
8 Ignition delay times of ethanol/DME/air mixtures were measured in a rapid compression machine and in two high-
9 pressure shock tubes at conditions relevant to internal combustion engines. The influences of these conditions on the
10 auto-ignition behavior of the mixture blends were systematically investigated. Our results indicate that, in the low
11 temperature range (650 – 950 K), increasing the amount of DME in the fuel mixture increases. At higher
12 temperatures, reactivity is controlled by ethanol and there is almost no visible impact of the fuel mixture
13 composition, whereas DME shows a slower reactivity. The experimental measurements were simulated using an
14 updated mechanism for ethanol which includes the latest experimental or theoretical work in the literature. Results
15 indicates that the model is in satisfactory agreement with all of the mixtures.

16 1. Introduction

17
18 Ethanol is considered to be a promising transportation biofuel due to its sustainability [1,2] and low soot
19 emissions [3,4] in internal combustion engine (ICE). In order to consider its potential use in real internal combustion
20 engines, it is important to investigate *a priori* how it will behave under practical engine relevant conditions. Ethanol
21 also has an impact on the reactivity of surrogate fuels [5]. Compared to mixtures made of primary reference fuels
22 (PRF), n-heptane and iso-octane, ethanol reduces the NTC (Negative Temperature Coefficient) behavior at low
23 temperature and increases the reactivity at intermediate and high temperature. Thus, ethanol has been previously
24 extensively studied under various range of conditions including flame speed measurements [6 - 12], ignition delay
25 times measurements in shock tube [13 - 17] and rapid compression machines [17 - 19], species measurements in
26 rapid compression facility [19], flow reactors [5,20,21] and jet-stirred reactors [22]. There is, however, limited
27 experimental data with which to evaluate the auto-ignition behavior of ethanol under engine relevant conditions:
28 high pressure and low to intermediate temperature (especially below 850 K) [18] since most of the experimental
29 investigation has been performed at low pressure (below 5 bar) or at high temperature (above 1000 K). Considering
30 measurements performed at high pressure, Heufer and Olivier [15] measured the ignition delay times of the

*Corresponding author: alan.keromnes@u-bourgogne.fr

31 stoichiometric ethanol-air mixture at 13, 19, 40 and 75 bar over a temperature range of 910 – 1190 K using a shock
32 tube. Cancino et al. [16] investigated the ignition of for lean ethanol-air mixture ($\phi = 0.3$) at 30 bar between 860 K
33 and 1180 K and for stoichiometric mixture over a pressure range of 10 to 50 bar with a temperature range of 800 K
34 to 1250 K in a shock tube. Lee et al. [17] performed a shock tube and RCM study of stoichiometric ethanol-air
35 mixture under high pressure conditions (67 – 93 bar) in the intermediate temperature range (775 – 1000 K). Mittal
36 et al. [18] investigated the auto-ignition behavior of lean ethanol-air mixture ($\phi = 0.3$ and 0.5) in the intermediate
37 temperature range (830 – 980 K) for a pressure range of 10-50 bar and of stoichiometric mixture at 10 bar and over
38 limited temperature range (870 – 920 K). More recently, Barraza-Botet et al. [19] measured ignition delay times and
39 species concentration for diluted stoichiometric ethanol/O₂ mixture in a rapid compression facility between 880 K
40 and 1120 K over a pressure range of 3 – 10 bar. Extended experimental work on ethanol ignition, including lean to
41 rich mixtures, has been mainly performed at high temperature and low pressure or diluted conditions in shock tubes
42 [13,23,24]. The available data are summarized in Figure 1. It appears that there is a lack of data at low temperature
43 and high pressure (Figure 1 Figure 1b). This lack of data limits the construction, validation and interpretation of chemical
44 kinetic mechanism for the real combustor design. Such data are particularly important in assessing the influence of
45 blending ethanol with practical fuels, in which the low reactivity of ethanol will inhibit the reactivity of a fuel at
46 these low temperature conditions [25,26]. Moreover, Mittal et al. [18] reveal some discrepancies in the data
47 available at low temperature (Figure 1 Figure 1a) whereas there is a very good agreement at high temperature. They attribute
48 part of this scatter to pre-ignition pressure rise in shock tubes. The recent study from Barraza-Botet et al. [19]
49 suggests that the accuracy of rate constants for ethanol + HO₂ needs to be improve. This is confirmed by Olm et al.
50 [27] who performed an optimization of ethanol mechanism previously published by Saxena and Williams [9].
51 Therefore, low temperature chemistry of ethanol might be questionable.

52 Detailed chemical kinetics mechanisms have been proposed by several authors [7,9,16,17,22,28–31]. Dunphy et
53 al. [28] validated a kinetic mechanism against ignition delay times measurements performed at low pressures (2-4.5
54 bar) and at high temperatures (1100 – 1500 K). Egolfopoulos et al. [7] proposed a mechanism tested under low
55 pressure conditions including ignition delay times, species profiles measured in flow reactor and laminar flame
56 speeds. Marinov [29] validated his mechanism against a wide range of experimental conditions including laminar
57 flame speeds, species profiles measured in jet-stirred reactor and flow reactor and ignition delay times. Li et al. [30]
58 studied the decomposition of ethanol in a flow reactor and proposed a rate constant for the decomposition reaction
59 $C_2H_5OH = C_2H_4 + H_2O$ in order to improve the mechanism previously developed by Marinov. Saxena and Williams
60 [9] proposed a new mechanism validated at low pressure against flame speeds, ignition delay times and species

61 profiles. Cancino et al. [16] built their mechanism based on that of Marinov and validated it against ignition delay
62 times under high pressure conditions (10 – 50 bar) for lean and stoichiometric mixtures. Leplat et al. [22] based their
63 mechanism by combining the mechanism developed by Marinov [29] and GRI Mech [32] with previously released
64 new rate constant. They increased the pressure range by testing the mechanisms against jet-stirred reactor and flame
65 speed measured at 10 bar reproduced ignition delay times and species profiles with a good agreement. Lee et al. [17]
66 revisited the mechanism proposed by Li et al. [30] and extended the validation to high pressure (80 bar) based on
67 ignition delay times measurements performed in high-pressure shock tube and rapid compression machine for
68 stoichiometric ethanol-air mixtures. Metcalfe et al. [31] proposed a C1–C2 mechanism validated against a wide
69 range of conditions including ethanol combustion targets.

70 In the present study, ignition delay times have been measured in rapid compression machine and in shock tubes
71 covering lean to rich conditions ($\phi = 0.5, 1.0$ and 2.0) over a wide range of temperature (650 – 1250 K) under
72 pressure conditions relevant to internal combustion engines (20 and 40 bar). However, due to limitations of
73 experimental facilities (especially heat transfer), it is difficult to measure long ignition delay times at temperature
74 below 825 K for ethanol-air mixture. Recently, in order to probe its low temperature chemistry, toluene was blended
75 with dimethyl ether (DME) in order to test a mechanism's ability to reproduce accurately the reactivity of various
76 binary mixture combinations [33]. Due to the success of that study, DME has been selected as a radical initiator
77 once again to test the predictive capability of a mechanism to accurately reproduce the experimentally observed
78 reactivity of binary ethanol/DME mixtures and extend the temperature range to 650 K. In order to assess the
79 inhibiting effect of ethanol, different blending ratios have been tested: ethanol/DME: 0/100, 50/50, 30/70, 100/0.
80 Using this experimental database, the chemical kinetic mechanism has been revised and its performance has been
81 evaluated against these binary mixtures and data available in the literature.

82 In the following sections, the experimental devices are described, followed by a presentation and a discussion of
83 the experimental results. Then, the revised chemical kinetic mechanism is presented and its performances in
84 reproducing experimental results is discussed.

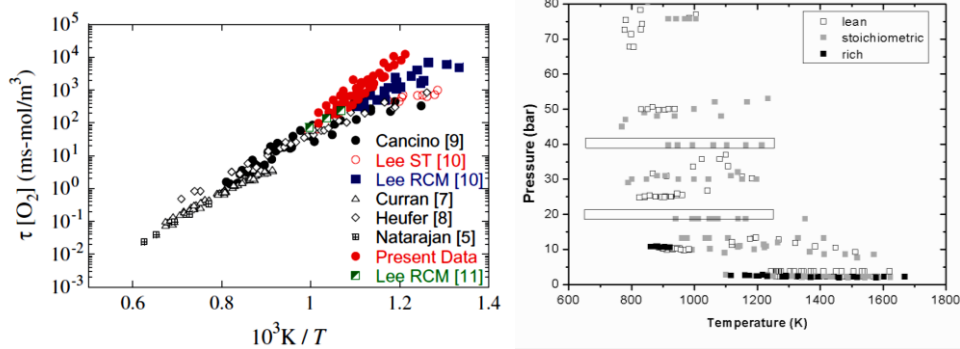


Figure 1: Comparison of the available ignition delay times data (symbols) with the experimental conditions presented in this study

85

86 2. Experimental devices

87 The experiments were carried out in a rapid compression machine (RCM) in NUI Galway and two high-pressure
 88 shock tubes (NUI Galway and DRIVE). Ignition delay times of ethanol/DME/air mixtures were measured at
 89 conditions relevant to those encountered in ICEs (pressure: 20 – 40 bar, temperature: 650 – 1250 K, equivalence
 90 ratio: 0.5 – 2.0, and blending ratio: ethanol/DME: 0/100, 50/50, 30/70, 100/0 in air). All mixtures were prepared
 91 manometrically in heated stainless steel tanks. The partial pressure of ethanol is maintained below one third of its
 92 vapor pressure in order to avoid any condensation of the fuels in the tanks. Moreover, all of the intake manifolds, the
 93 RCM and the two HPSTs are heated for the same purpose. Ethanol used in *this study* was from Sigma-Aldrich at
 94 99.5 +% purity and O₂, N₂ and He were supplied by BOC Ireland and Air Liquide at 99.5%, 99.95% and 99.9%,
 95 respectively.

96 The RCM is a horizontally-opposed twin-piston device that has been described previously [34,35]. The
 97 symmetry of the RCM allows a short adiabatic compression process (16 – 17 ms) and helps to reduce the
 98 aerodynamic effects inside the combustion chamber at the end of the compression process [36]. Moreover, the
 99 piston heads include a crevice shape which captures the vortex created by the piston corner compression. Thus, the
 100 mixture and the temperature are homogeneous prior to ignition. The pressures and temperatures achieved at the end
 101 of the compression process (p_C and T_C respectively) in this study were 20 and 40 bar and 650 – 1050 K, respectively.
 102 The final conditions were reached by changing the initial pressure and temperature. For all experiments, the pressure
 103 and the position of both pistons are recorded using a digital oscilloscope. The pressure profile recorded with a
 104 pressure transducer (Kistler 603B) provided the compression time and was used to measure the ignition delay time.

105 It was defined as the time difference between the end of the compression process and the maximum rate of increase
106 of the pressure. The temperature, T_c , is calculated using the initial temperature, T_i , and pressure, p_i , and the pressure,
107 p_c , assuming adiabatic compression and frozen chemistry and using Gaseq [37]. For each experimental conditions,
108 an experiment with a non-reactive mixture was performed by replacing oxygen by nitrogen in the test mixture since
109 they have similar thermodynamic properties. The recorded pressure profile is used in order to take into account heat
110 losses when simulating the experiments.

111 The experiments in DRIVE were carried out in a high-pressure shock tube which has been previously described
112 [38]. The stainless steel tube has an inner diameter of 50 mm and is divided by a double membrane (stainless steel
113 diaphragm) into two sections (a driver section of 4 m and a driven section of 5 m) constituting a small section called
114 “intermediate section (IS)”. In addition of the main tube, this facility includes a vacuum system (a roughing pump
115 and a turbo-molecular pump) which pumped down the tube and the stainless steel tanks to pressures below 5 Pa, a
116 velocity detection system (based on four individual piezoelectric pressure transducers PCB 113B22) and a data
117 acquisition system (NI Compact RIO). The tube, mixing tanks and manifold were pre-heated to 80°C to avoid any
118 condensation of ethanol by ensuring its partial pressure is below one third of its vapor pressure. Post-shock
119 pressures, p_s , are measured using a Kistler piezoelectric pressure transducer (603B1) located at the endwall. The
120 temperature, T_s , behind the reflected shock wave is calculated using the shock wave velocity in conjunction with the
121 1-D shock relations and the species thermodynamics using the chemical equilibrium software Gaseq [37] with an
122 accuracy of $\pm 1\%$ which corresponds to $\pm 10 - 15$ K. The Kistler pressure transducer is also used to determine the
123 qualitative, transient pressure and to determine the ignition delay time. It is defined as the time interval between the
124 time when the pressure and temperature conditions behind the reflected shock wave are reached and the onset of
125 combustion, commonly defined by sudden change in pressure.

126 Only ignition delay times of ethanol/air mixture at 20 and 40 atm at $\phi = 1.0$ were measured in the NUI Galway
127 HPST, described previously [39], in order to cross-check the reliability of the data recorded in both facilities.
128 Briefly, the HPST has a constant inner diameter of 63.5 mm with a 3.0 m driver section and a 5.7 m driven section.
129 Two pre-scored aluminum diaphragms were used to promote an ideal bursting and minimize undesirable fluid
130 dynamics during incident shock formation. The tube, mixing tank and manifold were pre-heated to 75 °C to allow
131 the partial pressure of ethanol in the prepared mixture was four times less than its vapor pressure. The prepared
132 mixtures were allowed to diffusively mix for 12 hours before performing experiments. The end-wall pressures,
133 monitored by a Kistler 603B pressure transducer, were used to identify the ignition delay times. The reflected shock
134 conditions were calculated using Gaseq [37] with input of the measured incident shock velocity determined by six

135 PCB 113B24 pressure transducers. The largest uncertainties presented here were estimated to be $\pm 15\%$ for ignition
 136 delay times and ± 20 K in reflected shock temperatures.

137 All mixtures were tested in the RCM in order to measure the ignition delay times in the low temperature range.
 138 For the experiments at higher temperatures, they were performed in the DRIVE high-pressure shock tube except the
 139 mixture without ethanol which were previously measured in NUIG [40]. However, in order to complete the
 140 database, ignition delay times of DME were measured in DRIVE at 40 bar and $\phi = 1$ and 2. Moreover, the
 141 experiments without DME were measured in both shock tubes to allow the comparison of experimental results and
 142 showed good agreement. The test matrix is detailed in [Table 1](#).

143 *Table 1: Mixtures investigated in this study*

Equivalence ratio (ϕ)	DME (%)	Ethanol (%)	O ₂ (%)	N ₂ (%)	Device
0.5	–	3.38	20.29	76.33	NUIG RCM & ST + DRIVE ST
	1.01	2.37	20.29	76.33	NUIG RCM + DRIVE ST
	1.69	1.69	20.29	76.33	NUIG RCM + DRIVE ST
	3.38	–	20.29	76.33	NUIG RCM & ST
1.0	–	6.54	19.63	73.83	NUIG RCM + DRIVE ST
	1.96	4.58	19.63	73.83	NUIG RCM + DRIVE ST
	3.27	3.27	19.63	73.83	NUIG RCM + DRIVE ST
	6.54	–	19.63	73.83	NUIG RCM + DRIVE ST
2.0	–	12.28	18.42	69.30	NUIG RCM + DRIVE ST
	3.68	8.60	18.42	69.30	NUIG RCM + DRIVE ST
	6.14	6.14	18.42	69.30	NUIG RCM + DRIVE ST
	12.28	–	18.42	69.30	NUIG RCM + DRIVE ST

144

145 3. Update of chemical kinetic mechanism

146 The base mechanism used in the current study is taken from AramcoMech 1.3, which includes the H₂/CO/O₂
 147 sub-mechanism developed by K romn s et al. [41] and the C1 – C2 sub-mechanism established by Metcalfe et
 148 al.[42]. The sub-mechanisms of ethanol and DME are adopted from the recent work of Mittal et al. [18] and Burke
 149 et al. [40] respectively. The original model shows an acceptable prediction at high temperatures whereas it appears
 150 to be unsatisfactory in simulating these new ignition delay time data measured in this study. Particularly for 50%
 151 EtOH/50% DME mixtures below 770 K, the mechanisms predicts a higher reactivity and ignition delay times which
 152 are up to 10-20% shorter than experimental results. This suggests that further improvement of the model of
 153 ethanol/DME binary fuel is warranted to refine the model’s ability to simulate the data over a wider range of
 154 conditions. The important reactions controlling both ethanol and DME oxidation chemistry are highlighted in the
 155 “brute-force” sensitivity analysis, Fig. 2, and will be discussed here.

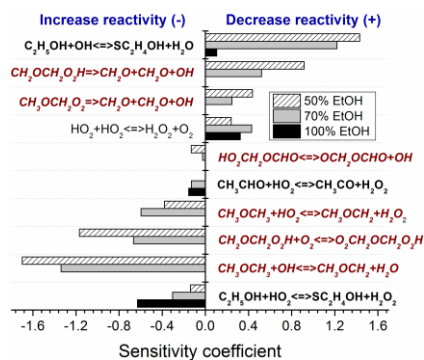


Figure 2: Brute-force sensitivity analysis to ignition delay times performed at $p = 20 \text{ atm}$, $\phi = 1.0$ and $T = 770 \text{ K}$ for 100% EtOH, and 70/30 and 50/50 EtOH/DME mixtures

3.1 H-atom abstraction

For the 100% EtOH system, H-atom abstraction by $\dot{H}O_2$ radicals at the alpha site on ethanol shows the highest promoting effect on ignition delay times. Unfortunately, neither experimental measurements nor theoretical calculations of this rate constant are so far available in the literature. Therefore, similar to the work of both Mittal et al. [18] and Metcalfe et al. [42], an analogical analysis of the reaction of *n*-butanol with $\dot{H}O_2$ radicals calculated by Zhou et al. [43] was made. Specifically, the total rate of $\dot{H}O_2$ radical consumption via H-atom abstractions was increased by a factor of 2 which is higher than the value stated by Mittal et al. (1.75) to better predict the reactivity of ethanol/air mixtures at high pressures (76 bar) reported in [15]. Moreover, the same branching ratio for alpha, beta and $\dot{O}H$ sites proposed by Zhou et al. was applied to estimate rates at the other two abstraction sites.

H-atom abstraction from ethanol by $\dot{O}H$ radicals at the alpha site shows the strongest inhibiting effect and the promotion is more prominent in the presence of DME due to the higher concentrations of $\dot{O}H$ radicals generated from the low temperature chain-branching processes involved in DME oxidation. This observation is similar to our recent study of toluene/DME ignition [33]. The rate constant used in Mittal et al was originally taken from Sivaramakrishnan et al. [44], but the authors increased the A-factor by 25% and maintained the total rate of $\dot{O}H$ consumption via H-atom abstraction from ethanol. However this adjustment leads to a relatively large branching ratio (over 90%) for this channel resulting in the prediction of increased concentrations of acetaldehyde and lower concentrations of ethylene compared to the data reported by Li et al. [45]. It is worth noting that the recommendation of Mittal et al. did not agree with the recent measurement (75 – 80% for the branching ratio) by Stranic et al. [46]. In this study, the total rate is adopted from the Stranic et al. measurement which is quite close to Sivaramakrishnan et al. Moreover, the rate constant of the channel forming SC_2H_4OH and H_2O was reduced by 30% to agree with the branching ratio measured by Stranic et al.

181 **3.2 Fuel radical decomposition**
 182 The reaction of acetaldehyde with HO₂ radicals *via* H-atom abstraction promotes reactivity for the 100% EtOH
 183 and 70% EtOH/30% DME systems. Mendes et al. [47] calculated the rate constant at the MP2/6-311G(d,p) level of
 184 theory combined with conventional transition state theory with an asymmetric Eckart tunneling correction. Their
 185 determined value is close to the recommendations of Baulch et al. [48] and da Silva and Bozzelli [49] and it is thus
 186 selected to describe this reaction.

187 It can be clearly seen that with increasing concentrations of DME, the reactions controlling ignition kinetic
 188 gradually transition to DME chemistry. The rate constants related to DME low temperature chemistry highlighted in
 189 Fig. 2 has been carefully optimized by Burke et al. [40]. However, the decomposition of the carbonyl-hydroperoxide
 190 (HO₂CH₂OCHO) forming $\dot{O}CH_2OCHO$ and $\dot{O}H$ radicals still has a large uncertainty. In this study, the rate constant
 191 is originally adopted from Sahetchian et al. [50] recommendation which has been used by Burke et al. [40], Curran
 192 et al. [51,52] and Zhao et al. [53], but these authors increased the rate constant by a factor of 5, 10 and 24. To
 193 achieve a reasonable adjustment, we only increase the rate constant of Sahetchian et al. by a factor of 2.5 which is
 194 within the uncertainty of their measurement.

195
 196 **3.3 Well-skipping reactions**
 197

198 **3.4 Waddington Mech and second O₂ addition**
 199

200 **3.5 Updated model performance**

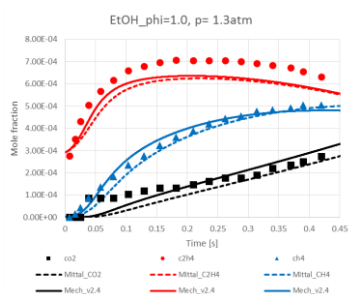


Figure 3: Ethanol FR data 1.3 atm, $\Phi = 1.3$, 950 K

201
 202

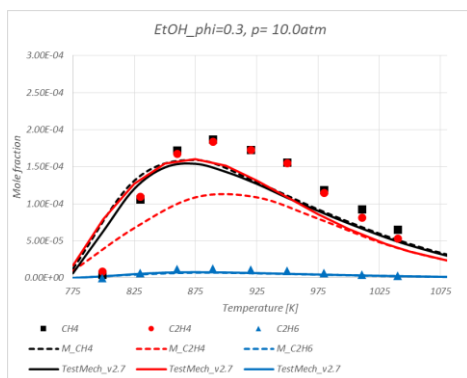


Figure 4: Ethanol JSR data 10 atm, $\Phi = 0.3$ [22]

203

204 A detailed comparison of the updated and original models compared to the new experimental data will be
 205 discussed in the following section.

206

207

208 4. Results and discussion

209 All mixtures were tested in the RCM in order to measure the ignition delay times in the low temperature range.
 210 For the experiments at higher temperatures, they were performed in the DRIVE-HPST except for the DME mixtures
 211 which were previously measured in the NUIG HPST [40]. Moreover, the 100% EtOH mixtures at 20 and 40 atm, at
 212 $\varphi = 1.0$ were studied in both facilities to allow an inter-comparison of experimental results, with good agreement
 213 observed.

214

215

215 4.1 Effect of physical conditions on reactivity of pure fuels

216 Generally, for the mixtures with a given blending ratio, the experimental trends with respect to the influence on
 217 ignition delay time of temperature, pressure and equivalence ratio are very much in-line chemical intuition and will
 218 not be discussed in great detail here. The extreme cases of 100% EtOH and 100% DME ignition will be briefly
 219 described to highlight the antagonistic behavior of EtOH on DME and/or the promoting effect of DME on EtOH and
 220 to provide a basis for discussion of the behaviors of the fuel blends. All of the measured and simulated ignition
 221 delay times are provided in Figs. 3–8.

222 For 100% EtOH the experimental temperature dependence can be essentially correlated using an Arrhenius or
 223 modified-Arrhenius type correlation in both the high- and low-temperature regimes without a change in the global
 224 activation energy. This is true for all conditions of pressure and equivalence ratio. Both the updated and original

225 mechanisms are capable of predicting the Arrhenius-type dependence with very little difference between them, as
226 under the conditions studied EtOH shows little or no reactivity at temperatures below 900 K. Hence our need to use
227 a highly-reactive radical initiator, DME, to induce low temperature chemistry in order to shorten ignition times and
228 permit the measurement of ignition delay times.

229 For 100% DME the ignition delay times show a straight-forward Arrhenius-type dependence on temperature
230 above 1050 K, whereas at temperatures below this a typical NTC behavior can be observed due to its well-
231 established chain-branching reaction mechanism. Again, both models accurately simulate this behavior.

232 It is clear that the original and updated models are valid in predicting ignition times for ethanol/air mixtures
233 above 800 K and DME/air mixtures over the entire temperature ranges investigated, for all conditions of pressures
234 and equivalence ratios. The fundamental question of this work now arises. Do the models retain their predictability
235 of the experimental measurements beyond the previous validation ranges studied?

236

237 *4.2 Effect of blending ratio on reactivity*

238 Our results indicate that, the addition of DME exhibits a two-fold effect on ethanol ignition: 1) DME inhibits the
239 reactivity of ethanol at higher temperatures ($T > 1050$ K) as ethanol undergoes either unimolecular decomposition or
240 beta-H atom abstraction to form the highly reactive species, ethylene and $\dot{\text{O}}\text{H}$ radicals, which result in accelerated
241 ignition. By contrast, only less reactive species, $\dot{\text{C}}\text{H}_3$ radicals and formaldehyde are formed in DME oxidation at
242 high temperature *via* either unimolecular decomposition or H-atom abstraction followed by C–O beta scission,
243 resulting in the inhibition of ignition; 2) DME promotes the reactivity of ethanol at lower temperatures ($T = 650 -$
244 950 K) where ethanol mainly undergoes alpha-H atom abstraction followed by O–H bond β -scission forming less-
245 reactive acetaldehyde while DME can undergo the low temperature chain-branching process to form abundant $\dot{\text{O}}\text{H}$
246 radicals. As a result, DME addition shortens the ignition delay times at lower temperatures, as shown in Figs. 3–8.

247 Interestingly, the 70% EtOH/30% DME mixtures retain the ignition behavior of ethanol suggesting that ethanol
248 chemistry dominates the ignition kinetics of these mixture even though more reactions involving DME chemistry
249 appear in the sensitivity analysis, Fig. 2. Both models reproduce well the experimental observations. For the 50%
250 EtOH/50% DME mixtures, the ignition behavior is closer to that of pure DME, showing NTC behavior. The original
251 model is capable of predicting the experimental data at temperatures above 910 K but it under-predicts the reactivity
252 at temperature below 910 K. Obviously, the modification in the reaction of $\text{HO}_2\text{CH}_2\text{OCHO}$ decomposition improves
253 agreement in terms of kinetic chemistry of the binary fuel mixtures.

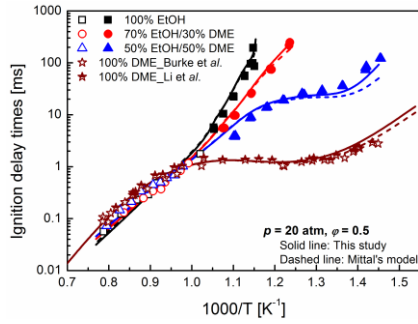


Figure 53: comparison of measured ignition delay times (open symbols: DRIVE-HPST, close symbols: RCM) with model predictions (lines) for 20 bar.

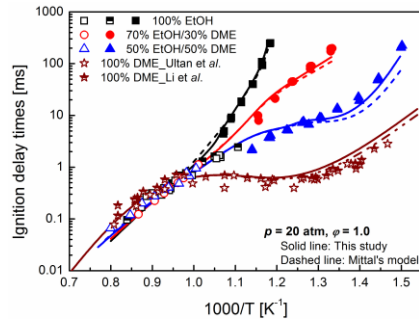


Figure 64: Comparison of measured ignition delay times (open symbols: DRIVE-HPST, close symbols: RCM, half-opened symbols: NUIG-HPST) with model predictions (lines) at $\phi = 1$ and 20 bar.

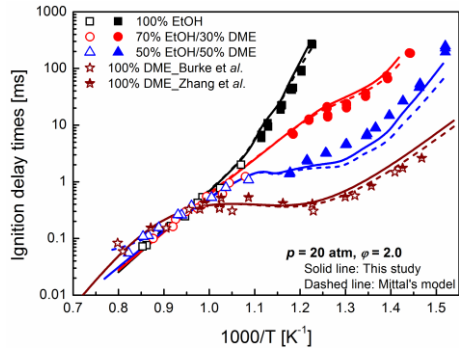


Figure 75: Comparison of measured ignition delay times (open symbols: DRIVE-HPST, close symbols: RCM) with model predictions (lines) at $\phi = 2$ and 20 bar.

267
268
269

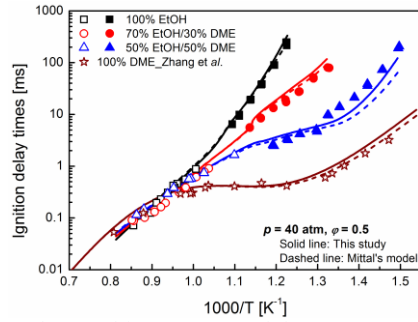


Figure 86: Comparison of measured ignition delay times (open symbols: DRIVE-HPST, close symbols: RCM) with model predictions (lines) at $\phi = 0.5$ and 40 bar.

270
271
272

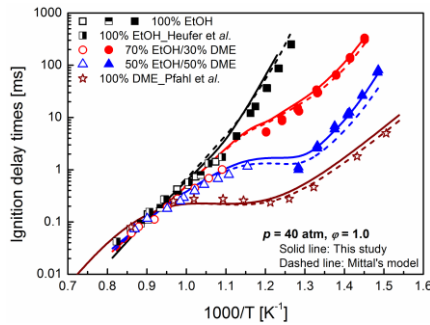


Figure 97: Comparison of measured ignition delay times (open symbols: DRIVE-HPST, close symbols: RCM, half-opened symbols: NUIG-HPST) with model predictions (lines) at $\phi = 1$ and 40 bar.

273
274
275

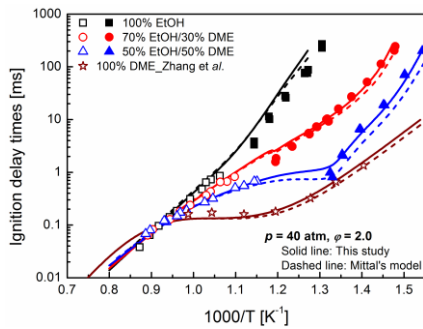


Figure 108: Comparison of measured ignition delay times (open symbols: DRIVE-HPST, close symbols: RCM, half-opened symbols: NUIG-HPST) with model predictions (lines) at $\phi = 1$ and 40 bar.

276

277

278 5. Conclusion

279

280

281

282

283 **Acknowledgments**

284 The DRIVE laboratory thanks French National Research Agency under the ANR project SHOCK (ANR-13-JS09-
285 0013-01) and the Council of Burgundy for their financial support.

286
287
288

References

- 289 [1] P.S. Nigam, A. Singh, Production of liquid biofuels from renewable resources, *Prog. Energy Combust. Sci.*
290 37 (2011) 52–68. doi:10.1016/j.pecs.2010.01.003.
- 291 [2] A. Demirbas, Progress and recent trends in biofuels, *Prog. Energy Combust. Sci.* 33 (2007) 1–18.
292 doi:10.1016/j.pecs.2006.06.001.
- 293 [3] V. Gargiulo, M. Alfè, G. Di Blasio, C. Beatrice, Chemico-physical features of soot emitted from a dual-fuel
294 ethanol–diesel system, *Fuel*. 150 (2015) 154–161. doi:10.1016/j.fuel.2015.01.096.
- 295 [4] M. Matti Maricq, Soot formation in ethanol/gasoline fuel blend diffusion flames, *Combust. Flame*. 159
296 (2012) 170–180. doi:10.1016/j.combustflame.2011.07.010.
- 297 [5] F.M. Haas, M. Chaos, F.L. Dryer, Low and intermediate temperature oxidation of ethanol and ethanol–PRF
298 blends: An experimental and modeling study, 2009. doi:10.1016/j.combustflame.2009.08.012.
- 299 [6] Ö.L. Gülder, Laminar burning velocities of methanol, ethanol and isooctane-air mixtures, *Symp. Combust.*
300 19 (1982) 275–281. doi:10.1016/S0082-0784(82)80198-7.
- 301 [7] F.N. Egolfopoulos, D.X. Du, C.K. Law, A study on ethanol oxidation kinetics in laminar premixed flames,
302 flow reactors, and shock tubes, *Symp. Combust.* 24 (1992) 833–841. doi:10.1016/S0082-0784(06)80101-3.
- 303 [8] T.S. Kasper, P. Oßwald, M. Kamphus, K. Kohse-Höinghaus, Ethanol flame structure investigated by
304 molecular beam mass spectrometry, *Combust. Flame*. 150 (2007) 220–231.
305 doi:10.1016/j.combustflame.2006.12.022.
- 306 [9] P. Saxena, F.A. Williams, Numerical and experimental studies of ethanol flames, *Proc. Combust. Inst.* 31
307 (2007) 1149–1156. doi:10.1016/j.proci.2006.08.097.
- 308 [10] K. Eisazadeh-Far, A. Moghaddas, J. Al-Mulki, H. Metghalchi, Laminar burning speeds of
309 ethanol/air/diluent mixtures, *Proc. Combust. Inst.* 33 (2011) 1021–1027. doi:10.1016/j.proci.2010.05.105.
- 310 [11] G. Broustail, P. Seers, F. Halter, G. Moréac, C. Mounaim-Rousselle, Experimental determination of laminar
311 burning velocity for butanol and ethanol iso-octane blends, *Fuel*. 90 (2011) 1–6.
312 doi:10.1016/j.fuel.2010.09.021.
- 313 [12] E. Varea, V. Modica, A. Vandel, B. Renou, Measurement of laminar burning velocity and Markstein length
314 relative to fresh gases using a new postprocessing procedure: Application to laminar spherical flames for
315 methane, ethanol and isooctane/air mixtures, *Combust. Flame*. 159 (2012) 577–590.
316 doi:10.1016/j.combustflame.2011.09.002.
- 317 [13] K. Natarajan, K.A. Bhaskaran, Experimental and analytical investigation of high temperature ignition of
318 ethanol, (1981).
- 319 [14] M.P. Dunphy, J.M. Simmie, High-temperature Oxidation of Ethanol Part 1.-Ignition Delays in Shock
320 Waves, *J. CHEM. SOC. FARADAY TRANS.* 87 (1991) 1691–1696.
- 321 [15] K.A. Heufer, H. Olivier, Determination of ignition delay times of different hydrocarbons in a new high
322 pressure shock tube, *Shock Waves*. 20 (2010) 307–316. doi:10.1007/s00193-010-0262-2.
- 323 [16] L.R. Cancino, M. Fikri, A.A.M. Oliveira, C. Schulz, Measurement and Chemical Kinetics Modeling of
324 Shock-Induced Ignition of Ethanol–Air Mixtures, *Energy & Fuels*. 24 (2010) 2830–2840.
325 doi:10.1021/ef100076w.
- 326 [17] C. Lee, S. Vranckx, K.A. Heufer, S. V. Khomik, Y. Uygun, H. Olivier, R.X. Fernandez, On the Chemical
327 Kinetics of Ethanol Oxidation: Shock Tube, Rapid Compression Machine and Detailed Modeling Study,
328 *Zeitschrift Für Phys. Chemie*. 226 (2012) 1–28. doi:10.1524/zpch.2012.0185.
- 329 [18] G. Mittal, S.M. Burke, V.A. Davies, B. Parajuli, W.K. Metcalfe, H.J. Curran, Autoignition of ethanol in a
330 rapid compression machine, *Combust. Flame*. 161 (2014) 1164–1171.
331 doi:10.1016/j.combustflame.2013.11.005.
- 332 [19] C.L. Barraza-Botet, S.W. Wagnon, M.S. Wooldridge, Combustion Chemistry of Ethanol: Ignition and
333 Speciation Studies in a Rapid Compression Facility, *J. Phys. Chem. A*. 120 (2016) 7408–7418.
334 doi:10.1021/acs.jpca.6b06725.
- 335 [20] F.Z. Li, D.K.S. Chen, SAE TECHNICAL The Solution for Steady State Temperature Distribution in
336 Monolithic Catalytic Converters, (2001).
- 337 [21] F. Herrmann, B. Jochim, P. Oßwald, L. Cai, H. Pitsch, K. Kohse-Höinghaus, Experimental and numerical
338 low-temperature oxidation study of ethanol and dimethyl ether, *Combust. Flame*. 161 (2014) 384–397.
339 doi:10.1016/j.combustflame.2013.09.014.
- 340 [22] N. Leplat, P. Dagaut, C. Togbé, J. Vandooren, Numerical and experimental study of ethanol combustion and

341 oxidation in laminar premixed flames and in jet-stirred reactor, *Combust. Flame*. 158 (2011) 705–725.
342 doi:10.1016/j.combustflame.2010.12.008.

343 [23] K.E. Noorani, B. Akih-Kumgeh, J.M. Bergthorson, Comparative High Temperature Shock Tube Ignition of
344 C1–C4 Primary Alcohols, *Energy & Fuels*. 24 (2010) 5834–5843. doi:10.1021/ef1009692.

345 [24] F.R. Gillespie, An experimental and modelling study of the combustion of oxygenated hydrocarbons, NUI
346 Galway, 2014.

347 [25] D.C. Rakopoulos, C.D. Rakopoulos, R.G. Papagiannakis, D.C. Kyritsis, Combustion heat release analysis of
348 ethanol or n-butanol diesel fuel blends in heavy-duty DI diesel engine, *Fuel*. 90 (2011) 1855–1867.
349 doi:10.1016/j.fuel.2010.12.003.

350 [26] S. Padala, C. Woo, S. Kook, E.R. Hawkes, Ethanol utilisation in a diesel engine using dual-fuelling
351 technology, *Fuel*. 109 (2013) 597–607. doi:10.1016/j.fuel.2013.03.049.

352 [27] C. Olm, T. Varga, É. Valkó, S. Hartl, C. Hasse, T. Turányi, Development of an Ethanol Combustion
353 Mechanism Based on a Hierarchical Optimization Approach, *Int. J. Chem. Kinet.* 48 (2016) 423–441.
354 doi:10.1002/kin.20998.

355 [28] M.P. Dunphy, P.M. Patterson, J.M. Simmie, High-temperature oxidation of ethanol. Part 2.—Kinetic
356 modelling, *J. Chem. Soc., Faraday Trans.* 87 (1991) 2549–2559. doi:10.1039/FT9918702549.

357 [29] N.M. Marinov, A detailed chemical kinetic model for high temperature ethanol oxidation, *Int. J. Chem.*
358 *Kinet.* 31 (1999) 183–220. doi:10.1002/(SICI)1097-4601(1999)31:3<183::AID-KIN3>3.0.CO;2-X.

359 [30] * Juan Li, and Andrei Kazakov, F.L. Dryer, Experimental and Numerical Studies of Ethanol Decomposition
360 Reactions, (2004). doi:10.1021/JP0480302.

361 [31] W.K. Metcalfe, S.M. Burke, S.S. Ahmed, H.J. Curran, A Hierarchical and Comparative Kinetic Modeling
362 Study of C₁ – C₂ Hydrocarbon and Oxygenated Fuels, *Int. J. Chem. Kinet.* 45 (2013) 638–675.
363 doi:10.1002/kin.20802.

364 [32] G.P. Smith, D.M. Golden, M. Frenklach, N.W. Moriarty, B. Eiteneer, M. Goldenberg, T. Bowman, R.K.
365 Hanson, S. Song, J. W.C. Gardiner, V. Lissianski, Z. Qin, http://www.me.berkeley.edu/gri_mech/, (n.d.).

366 [33] Y. Zhang, K.P. Somers, M. Mehl, W.J. Pitz, R.F. Cracknell, H.J. Curran, Probing the Antagonistic Effect of
367 Toluene as a Component in Surrogate Fuel Models at Low Temperatures and High Pressures. A Case Study
368 of Toluene/Dimethyl Ether Mixtures, in: *Proc. Combust. Inst.*, 2017: p. in press.

369 [34] L. Brett, J. MacNamara, P. Musch, J.M. Simmie, Simulation of methane autoignition in a rapid compression
370 machine with creviced pistons, *Combust. Flame*. 124 (2001) 326–329.

371 [35] S.M. Gallagher, H.J. Curran, W.K. Metcalfe, D. Healy, J.M. Simmie, G. Bourque, A rapid compression
372 machine study of the oxidation of propane in the negative temperature coefficient regime, *Combust. Flame*.
373 153 (2008) 316–333.

374 [36] J. Würmel, J.M. Simmie, CFD studies of a twin-piston rapid compression machine, *Combust. Flame*. 141
375 (2005) 417–430.

376 [37] C. Morley, Gaseq version 0.79, <Http://www.gaseq.co.uk/>. (2005).

377 [38] H. El Merhubi, A. Kéromnès, G. Catalano, B. Lefort, L. Le Moine, A high pressure experimental and
378 numerical study of methane ignition, *Fuel*. 177 (2016) 164–172. doi:10.1016/j.fuel.2016.03.016.

379 [39] D. Darcy, C.J. Tobin, K. Yasunaga, J.M. Simmie, J. Würmel, W.K. Metcalfe, T. Niass, S.S. Ahmed, C.K.
380 Westbrook, H.J. Curran, A high pressure shock tube study of n-propylbenzene oxidation and its comparison
381 with n-butylbenzene, *Combust. Flame*. 159 (2012) 2219–2232. doi:10.1016/j.combustflame.2012.02.009.

382 [40] U. Burke, K.P. Somers, P. O’Toole, C.M. Zinner, N. Marquet, G. Bourque, E.L. Petersen, W.K. Metcalfe,
383 Z. Serinyel, H.J. Curran, An ignition delay and kinetic modeling study of methane, dimethyl ether, and their
384 mixtures at high pressures, *Combust. Flame*. 162 (2015) 315–330. doi:10.1016/j.combustflame.2014.08.014.

385 [41] A. Kéromnès, W.K. Metcalfe, K.A. Heufer, N. Donohoe, A.K. Das, C.-J.J. Sung, J. Herzler, C. Naumann, P.
386 Griebel, O. Mathieu, M.C. Krejci, E.L. Petersen, W.J. Pitz, H.J. Curran, An experimental and detailed
387 chemical kinetic modeling study of hydrogen and syngas mixture oxidation at elevated pressures, *Combust.*
388 *Flame*. 160 (2013) 995–1011. doi:10.1016/j.combustflame.2013.01.001.

389 [42] W.K. Metcalfe, S.M. Burke, S.S. Ahmed, H.J. Curran, A Hierarchical and Comparative Kinetic Modeling
390 Study of C₁ – C₂ Hydrocarbon and Oxygenated Fuels, *Int. J. Chem. Kinet.* 45 (2013) 638–675.
391 doi:10.1002/kin.20802.

392 [43] C.-W. Zhou, J.M. Simmie, H.J. Curran, Rate constants for hydrogen abstraction by HO₂ from n-butanol, *Int.*
393 *J. Chem. Kinet.* 44 (2012) 155–164. doi:10.1002/kin.20708.

394 [44] R. Sivaramakrishnan, M.-C. Su, J. V. Michael, S.J. Klippenstein, L.B. Harding, B. Ruscic, Rate Constants
395 for the Thermal Decomposition of Ethanol and Its Bimolecular Reactions with OH and D: Reflected Shock
396 Tube and Theoretical Studies, *J. Phys. Chem. A*. 114 (2010) 9425–9439. doi:10.1021/jp104759d.

397 [45] J. Li, Z.W. Zhao, A. Kazakov, M. Chaos, F.L. Dryer, J.J. Scire, A comprehensive kinetic mechanism for
398 CO, CH₂O, and CH₃OH combustion, *Int. J. Chem. Kinet.* 39 (2007) 109–136. doi:10.1002/kin.20218.

399 [46] I. Stranic, G.A. Pang, R.K. Hanson, D.M. Golden, C.T. Bowman, Shock Tube Measurements of the Rate
400 Constant for the Reaction Ethanol + OH, *J. Phys. Chem. A*. 118 (2014) 822–828. doi:10.1021/jp410853f.

- 401 [47] J. Mendes, C.-W. Zhou, H.J. Curran, Theoretical Chemical Kinetic Study of the H-Atom Abstraction
402 Reactions from Aldehydes and Acids by $\dot{\text{H}}$ Atoms and $\dot{\text{O}}\text{H}$, $\text{H}\dot{\text{O}}_2$, and $\dot{\text{C}}\text{H}_3$ Radicals, *J. Phys. Chem. A*. 118
403 (2014) 12089–12104. doi:10.1021/jp5072814.
- 404 [48] D.L. Baulch, C.J. Cobos, R.A. Cox, C. Esser, P. Frank, T. Just, J.A. Kerr, M.J. Pilling, J. Troe, R.W.
405 Walker, J. Warnatz, Evaluated kinetic data for combustion modeling, *J. Phys. Chem. Ref. Data*. 21 (1992)
406 411–734.
- 407 [49] G. da Silva, J.W. Bozzelli, Role of the α -hydroxyethylperoxy radical in the reactions of acetaldehyde and
408 vinyl alcohol with HO_2 , *Chem. Phys. Lett.* 483 (2009) 25–29. doi:10.1016/j.cplett.2009.10.045.
- 409 [50] K.A. Sahetchian, R. Rigny, J. Tardieu de Maleissye, L. Batt, M. Anwar Khan, S. Mathews, The pyrolysis of
410 organic hydroperoxides (ROOH), *Symp. Combust.* 24 (1992) 637–643. doi:10.1016/S0082-0784(06)80078-
411 0.
- 412 [51] S.L. Fischer, F.L. Dryer, H.J. Curran, The reaction kinetics of dimethyl ether. I: High-temperature pyrolysis
413 and oxidation in flow reactors, *Int. J. Chem. Kinet.* 32 (2000) 713–740. doi:10.1002/1097-
414 4601(2000)32:12<713::AID-KIN1>3.0.CO;2-9.
- 415 [52] H.J. Curran, S.L. Fischer, F.L. Dryer, The reaction kinetics of dimethyl ether. II: Low-temperature oxidation
416 in flow reactors, *Int. J. Chem. Kinet.* 32 (2000) 741–759. doi:10.1002/1097-4601(2000)32:12<741::AID-
417 KIN2>3.0.CO;2-9.
- 418 [53] Z. Zhao, M. Chaos, A. Kazakov, F.L. Dryer, Thermal decomposition reaction and a comprehensive kinetic
419 model of dimethyl ether, *Int. J. Chem. Kinet.* 40 (2008) 1–18. doi:10.1002/kin.20285.
- 420